

# # Problem Session 4

2/3/2025

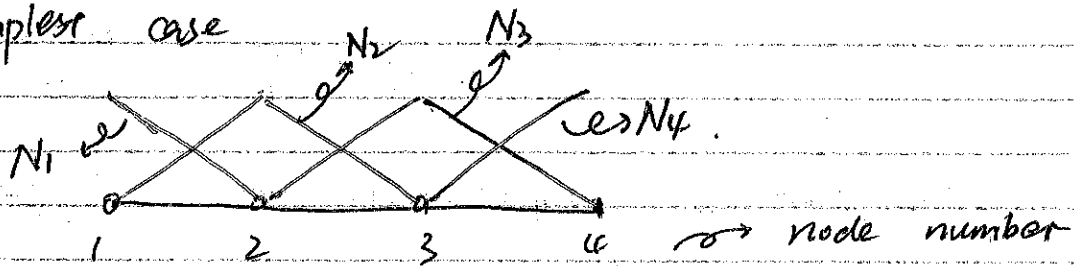
▷ Local to Global Map

▷ Examples on FEM implementation

▷ R & A.

## Local to Global Map

Simplest case



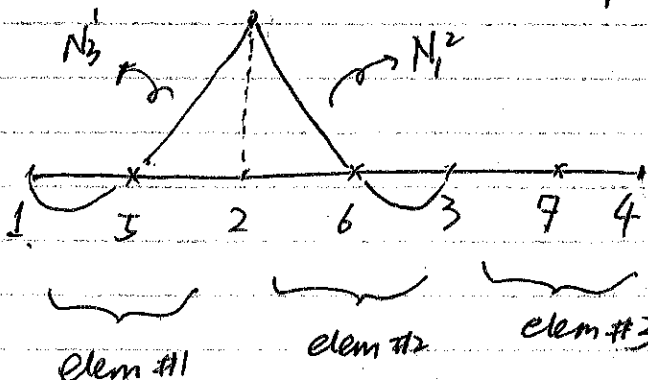
$$LG = \begin{bmatrix} 1 & 2 & 3 \\ 2 & 3 & 4 \\ 2 & 3 & 4 \end{bmatrix}$$

$\underbrace{\quad\quad\quad}_{\text{element \#1}} \quad \underbrace{\quad\quad\quad}_{\text{elem. \#2}} \quad \underbrace{\quad\quad\quad}_{\text{elem. \#3}}$

$$LG = \begin{bmatrix} 1 & 2 & 3 \\ 5 & 6 & 7 \\ 2 & 2 & 4 \end{bmatrix}$$

### # Example 3.11

continuous piecewise quadratic functions



$$N_1 = N_1^1$$

$$N_2 = N_3^1 + N_1^2$$

$$N_3 = N_3^2 + N_1^3$$

$$N_4 = N_3^3$$

$$N_5 = N_2^1$$

$$N_6 = N_2^2, \quad N_7 = N_5^3$$

$N_a^b \rightarrow b$ : element number  
 $N_a^b \rightarrow a$ : shape functions (local)

Modified example from Philip DePond

▷ Consider a 1D diffusion-advection equation,

given constant  $k < 0$ ,  $f$ ,  $v$ .

... find  $T$  smooth enough s.t.

$$k \frac{d^2 T}{dx^2} + v \frac{dT}{dx} = f \quad \text{in } \Omega \in [-1, 1].$$

$$T(x=-1) = T_1$$

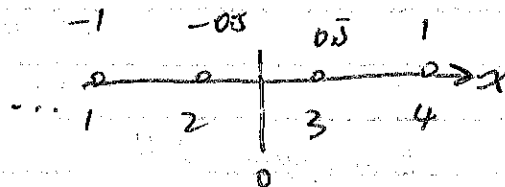
$$T(x=1) = T_2$$

B.C.s.

Consider a simple mesh w/ 4 nodes

using linear elements

Node	coordinate
1	-1
2	-0.5
3	0.5
4	1



Start Galerkin form:

Starting from strong form:

$$\int_{\Omega} \left( k \frac{d^2 T}{dx^2} + v \frac{dT}{dx} \right) w dx = \int_{\Omega} f w dx$$

$$\int T'' w dx = \int w dT'$$

$$\int_{\Omega} (T' w)' dx = \int T' w' dx$$

$$-k \int_{\Omega} \frac{dT}{dx} \cdot \frac{dw}{dx} dx + \nu \int_{\Omega} \frac{dT}{dx} w dx = \int_{\Omega} f w dx$$

the Galerkin form is stated:

$$a(w, T) = \int_{\Omega} \left( k \frac{dw}{dx} \frac{dT}{dx} - \nu \frac{dT}{dx} w \right) dx$$

$$l(w) = - \int_{\Omega} f w dx$$

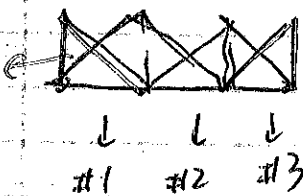
Find  $T_h \in \mathcal{T}_h = \text{span} \{N_1, N_2, N_3, N_4\}$  s.t.

$$a(w_h, T_h) = l(w_h) - a(w_h, T_h^g),$$

$$\forall w_h \in W_h = \mathcal{T}_h$$

~ Determine the LG matrix.

Impose  
B.C.s



$$LG = \begin{bmatrix} 1 & 2 & 3 \\ 2 & 3 & 4 \end{bmatrix}$$

Global statement for the finite element problem

$$\text{Dirichlet B.C.s: } T_h^g = T_1 N_1 + T_4 N_4.$$

$$\text{unknown part: } T_h = T_2 N_2 + T_3 N_3$$

$$\text{Full sol'n: } T_h^{\text{total}} = T_h^g + T_h = \sum_{i=1}^4 T_i N_i$$

test function  $w_h = \sum_{i=1}^4 w_i N_i$

$w_1 = w_4 = 0$  (due to Dirichlet B.C.s)

Recall def'n of bilinear func.

$a(u, w+v) = a(u, w) + a(u, v)$

Substitute into bilinear form:

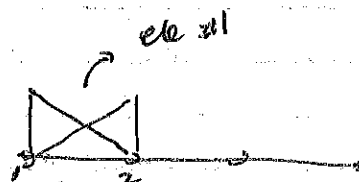
$$\sum_{i=1}^4 \sum_{j=1}^4 w_i a(N_i, N_j) T_j = \sum_{i=1}^4 w_i l(N_i) - \sum_{i=1}^4 w_i a(N_i, T_h^g)$$

Since  $w_1 = w_4 = 0$ , system reduces to

$$\sum_{j=2}^3 a(N_i, N_j) T_j = l(N_i) - a(N_i, T_h^g) \quad i=2,3$$

Local version of Finite Element.

Element #1



$$K_{ab}^1 = \int_{\Omega^1} \left( k \frac{dN_a^1}{dx} \frac{dN_b^1}{dx} - \nu \frac{dN_b^1}{dx} N_a^1 \right) dx, \quad a, b = 1, 2$$

$$F_a^1 = - \int_{\Omega^1} f N_a^1 dx - a(N_a^1, T_h^g), \quad a = 1, 2$$

$K^1, F^1$  corresponding to  $LG(a, 1)$  &  $LG(b, 1)$

Element #2

(nodes #2 & #3)

$$K_{ab}^2 = \int_{\Omega^2} \left( k \frac{dN_a^2}{dx} \frac{dN_b^2}{dx} - \nu \frac{dN_b^2}{dx} N_a^2 \right) dx, \quad a, b = 1, 2$$

$$F_a^2 = - \int_{\Omega^2} f N_a^2 dx - a(N_a^2, T_h^g), \quad a = 1, 2$$

$K^2, F^2$  correspond to  $LG(a, 2)$  &  $LG(b, 2)$

... same procedure with elements #3 & #4.

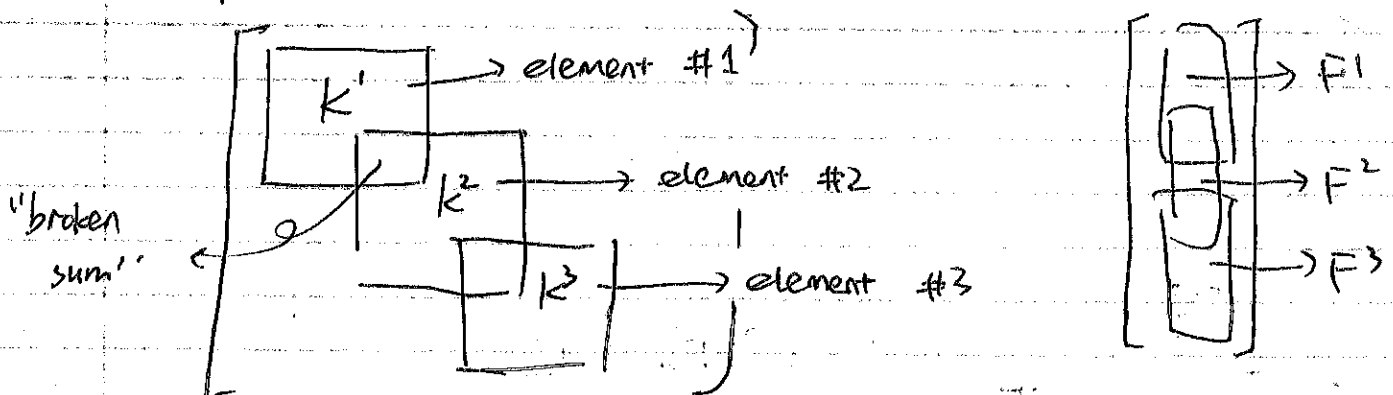
↳ corresponding to  $LG(a, 3), LG(b, 3) \dots ?$   
 $LG(a, 4), LG(b, 4)$

(Important !!) Assemble the Global System.

$$K_{LG(a,e), LG(b,e)} \leftarrow K_{LG(a,e), LG(b,e)} + K_{ab}^e \quad \text{for all } e, a, b$$

$$F_{LG(a,e)} \leftarrow F_{LG(a,e)} + F_a^e$$

Stiffness matrix



Final step: Solve the global system

$$KT = F$$

$$T = K^{-1}F$$

after solving for  $T$ , the sol'n:

$$T_h = T_1 N_1 + T_2 N_2 + T_3 N_3 + T_4 N_4$$

... implement these in

Python / MATLAB

~ your first FEM code!